

Scoring Indicators

COURSE NAME : Design of Machine Elements

COURSE CODE : 5021

QID : 2109230031 (A)

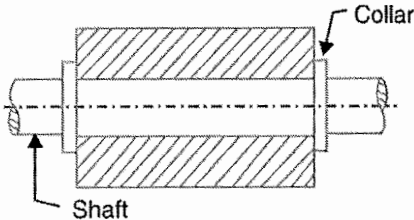
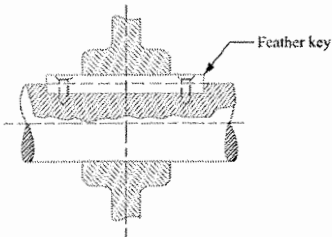
PART A

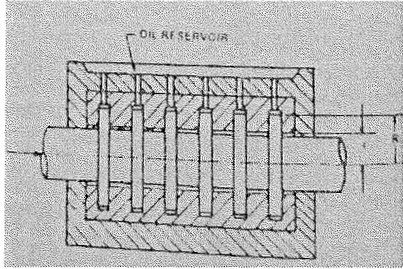
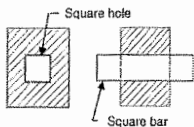
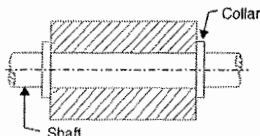
I. Answer all the following questions in one word or sentence.

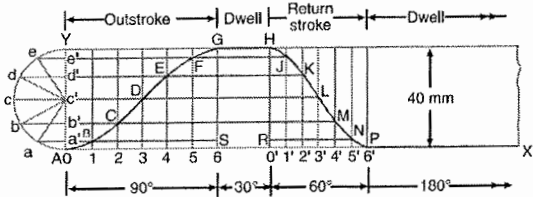
(9 x 1 = 9 Marks)

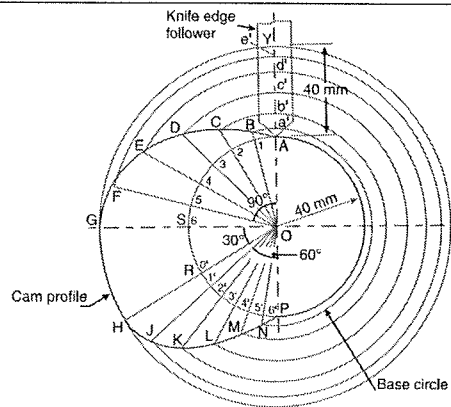
Max. marks

Q.No	Scoring Indicators	Split score	Sub Total	Total score
	PART A			24
I.1	Strength, cost, shape, size, safety, weight (any two)	0.5×2	1	
I.2	Each part of a machine, which moves relative to some other part, is known as a kinematic link (or simply link) or element. A link may consist of several parts, which are rigidly fastened together, so that they do not move relative to one another. (any one sentence)	1	1	
I.3	Transmission shaft, machine shaft	0.5×2	1	
I.4	Woodruff key	1	1	
I.5	It is a reference point on the follower and is used to generate the pitch curve.	1	1	
I.6	Internal combustion engines, press work	0.5×2	1	
I.7	Sometimes, the frictional grip becomes insufficient in the belt and pulley, cause some forward motion of the belt without carrying the driven pulley with it. This is called slip of the belt and is generally expressed as a percentage.	1	1	
I.8	It is the surface of the gear tooth below the pitch surface.	1	1	
I.9	They give smooth, steady and quiet service. They are little affected by outdoor conditions. The shafts may be out of strict alignment. The power may be taken off in any direction and in fractional parts of the whole amount. They give high mechanical efficiency. (any two)	0.5×2	1	
	PART B			24
II.1	When the motion between a pair is limited to a definite direction irrespective of the direction of force applied, then the motion is said to be a completely constrained motion. Eg: piston and cylinder, shaft with both side	Explanat ion – 1 Figure -	3	

	collar's.	2		
				
II.2	<p>Standard Sizes: 25mm to 60mm with 5mm steps.</p> <p>60mm to 110mm with 10mm steps.</p> <p>110mm to 140mm with 15mm steps.</p> <p>140mm to 500mm with 20mm steps. (any two)</p> <p>Standard length: 5m, 6m, 7m</p>	2 1	3	
II.3	<p>Power , $P = (2\pi NT)/60$</p> <p>Where N = Speed of shaft in rpm</p> <p>T = torque transmitted</p>	2 1	2	
II.4	<p>For solid shaft, $I = \frac{\pi}{32} d^4$</p> <p>For hollow shaft, $I = \frac{\pi}{32} (d_o^4 - d_i^4)$</p> <p>Where, d_o – outer diameter of shaft</p> <p>d_i - internal diameter of shaft</p>	1 1	3	
II.5	<p>A key attached to one member of the connected torque transmitting members and which permits relative axial movement is known as feather key. It is a special type of parallel key which transmits a turning moment and also permits axial movement. It is fastened either to the shaft or hub.</p> 	<p>Explanat ion – 1</p> <p>Figure - 2</p>	3	
II.6	<p>Wedge cam</p> <p>radial (disc) cam</p> <p>cylindrical cam</p>	1 × 3	3	

II.7	<p>Two types</p> <p>i. Centrifugal governors (further divided in to pendulum type, loaded type)</p> <p>ii. Inertia governors</p>	2 1	3	
II.8	<p>Rigid coupling(sleeve or muff type, flange type)</p> <p>Flexible coupling(bushed type, universal type)</p>	2 1	3	
II.9	<p>In this case a shaft passes through the bearings, bearing take up the axial loads, shaft having multiple collars.</p> 	<p>Explanat ion – 1</p> <p>Figure - 2</p>	3	
II.10	<p>Based on friction - belt drives, rope drives</p> <p>Based on engagement - chain drives, gear drives</p>	2 1	3	
II	PART C			42
<u>III</u>	<p>Sliding pair.</p> <p>When the two elements of a pair are connected in such a way that one can only slide relative to the other, the pair is known as a sliding pair. The piston and cylinder, cross-head and guides of a reciprocating steam engine, ram and its guides in shaper, tail stock on the lathe bed etc. are the examples of a sliding pair. A little consideration will show, that a sliding pair has a completely constrained motion.</p>  <p>Turning pair.</p> <p>When the two elements of a pair are connected in such a way that one can only turn or revolve about a fixed axis of another link, the pair is known as turning pair. A shaft with collars at both ends fitted into a circular hole, the crankshaft in a journal bearing in an engine, lathe spindle supported in head stock, cycle wheels turning over their axles etc. are the examples of a turning pair. A turning pair also has a completely constrained motion.</p> 	<p>Explanat ion – 3</p> <p>Figure – 4</p>	7	

	and hence in total there are four rows of rivets in double riveted joint.		
V/II	<p>Given:</p> <p>$d=50 \text{ mm}$, $\tau=42 \text{ MPa} = 42 \text{ N/mm}^2$</p> <p>$w=16 \text{ mm}$</p> <p>$t= 10 \text{ mm}$</p> <p>Considering shearing of the key,</p> $T= l \times w \times \tau \times d/2 = 16800 \text{ l Nmm} \quad \text{Equn. (1)}$ <p>Torque transmitted by the shaft,</p> $T= (\pi/16) \times \tau \times d^3 \quad \text{Equn. (2)}$ $= 1.03 \times 10^6 \text{ Nmm}$ <p>Equating 1 & 2,</p> <p>We have , $l= 61.31 \text{ mm}$.</p> <p style="text-align: center;">OR</p>	<p>Data 1</p> <p>2</p> <p>1</p> <p>1</p> <p>1</p>	7
VIII/8	<p>Given:</p> <p>$N=200 \text{ rpm}$, $\tau=42 \text{ MPa} = 42 \text{ N/mm}^2$</p> <p>$P=20 \text{ KW} = 20 \times 10^3 \text{ W}$</p> <p>$d=$ diameter of shaft</p> <p>Torque transmitted by the shaft,</p> $T= P \times 60 / (2\pi N) = 955 \times 10^3 \text{ Nmm}$ <p>Also, Torque transmitted by the shaft,</p> $T= (\pi/16) \times \tau \times d^3$ $955 \times 10^3 = 8.25 d^3$ <p>$d= 48.7 \text{ mm} = 50 \text{ mm}$</p>	<p>Data 1</p> <p>2</p> <p>1</p> <p>1</p> <p>1</p>	7
IX		<p>Displacement diagram -2</p> <p>Cam profile -5</p>	7



OR

	Flywheel	Governor
1.	Reduce the cyclic variation of speed due to difference in input and output.	Maintain a constant speed of prime mover when the variation of speed occurs due to fluctuation of load.
2.	Action is continuous from cycle to cycle.	Action is intermittent i.e. when load changes on engine.
3.	Stores energy and gives up whenever required during the cycle.	Regulates input energy (i.e. working fluid) according to the load so as to maintain constant speed.
4.	No control over the quality of working agent.	Changes the quality of the working agent.
5.	Not an essential element of every prime mover.	Essential element of every prime mover.

7

7

The rope drives are widely used where a large amount of power is to be transmitted, from one pulley to another, over a considerable distance. It may be noted that the use of flat belts is limited for the transmission of moderate power from one pulley to another when the two pulleys are not more than 8 metres apart. If large amounts of power are to be transmitted by the flat belt, then it would

result in excessive belt cross-section. The frictional grip in case of rope drives is more than that in V-drive. One of the main advantage of rope drives is that a number of separate drives may be taken from the one driving pulley. For example, in many spinning mills, the line shaft on each floor is driven by ropes passing directly from the main engine pulley on the ground floor.

The rope drives use the following two types of ropes :

1. Fibre ropes, and 2. Wire ropes.

OR

Given:

$N_1 = 150 \text{ rpm}$, $d_1 = 750 \text{ mm}$, $d_2 = 450 \text{ mm}$, $d_3 = 900 \text{ mm}$,
 $d_4 = 150 \text{ mm}$

5

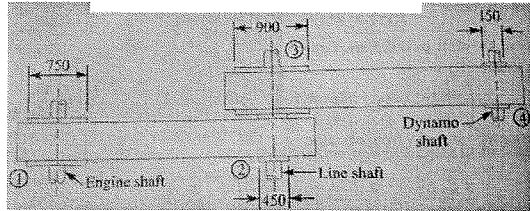
7

2

Data 1

7

N_4 = speed of the dynamo shaft



$$N_4 / N_1 = (d_1 \times d_3) / (d_2 \times d_4)$$

$$= (750 \times 900) / (450 \times 150)$$

$$N_4 = 1500 \text{ rpm.}$$

1

2

2

1

Advantages

As no slip takes place during chain drive

Perfect velocity ratio is obtained

Since the chains are made of metal, therefore they occupy less space in width than a belt drive.

The chain drives may be used when the distance between the shafts is less.

The chain drive gives a high transmission efficiency (upto 98 per cent).

The chain drive gives less load on the shafts.

The chain drive has the ability of transmitting motion to several shafts by one chain only. (ANY FIVE)

Disadvantages

The production cost of chains is relatively high.

The chain drive needs accurate mounting and careful maintenance.

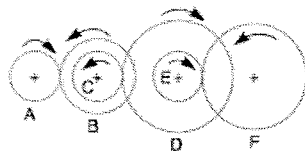
The chain drive has velocity fluctuations especially when unduly stretched. (ANY TWO)

1 × 5

7

1 × 2

OR



Gear	A	B	C	D	E	F
No. of teeth	20	50	25	75	26	65

N_F = speed of gear F (last driven follower)

$$(N_A / N_F) = (T_B \times T_D \times T_F) / (T_A \times T_C \times T_E)$$

$$(N_A / N_F) = (50 \times 75 \times 65) / (20 \times 25 \times 26)$$

$$N_F = 52 \text{ rpm}$$

Data 1

7

3

2

1